

# CBCS SCHEME

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21AU51

Fifth Semester B.E. Degree Examination, Dec.2023/Jan.2024

## Heat and Mass Transfer

Time: 3 hrs.

Max. Marks: 100

- Note: 1. Answer any FIVE full questions, choosing ONE full question from each module.  
2. Use of Thermodynamics, Heat transfer Data hand book is permitted.

### Module-1

- 1 a. Discuss the three basic modes of heat transfer. Write the governing equations for the same and the governing laws. (12 Marks)  
b. The heat flow rate across an insulating material of thickness 30 mm with thermal conductivity  $0.1 \text{ W/m}^\circ\text{C}$  is  $250 \text{ W/m}^2$ . If the hot surface is  $175^\circ\text{C}$ , find the temperature of the cold surface. (08 Marks)

OR

- 2 a. Derive one dimensional heat conduction equation for rectangular or Cartesian co-ordinates, in differential form. (10 Marks)  
b. An aircraft heat exchanger has a maximum wall temperature of  $810 \text{ K}$ . The hot and cold side heat transfer coefficients are respectively  $200 \text{ W/m}^2\text{K}$  and  $400 \text{ W/m}^2\text{K}$ . Find the maximum possible unit thermal resistance per  $\text{m}^2$  area of the metallic wall separating the hot gas from the cold gas, if the hot gas temperature is  $1200 \text{ K}$  and the coolant temperature is  $300 \text{ K}$ . (10 Marks)

### Module-2

- 3 a. Define :  
(i) Fin efficiency  
(ii) Fin effectiveness using the basic definition, arrive at the efficiency for,  
• a long fin of rectangular cross section  
• rectangular fin with insulated tip. (10 Marks)  
b. Calculate the heat loss rate from a rectangular fin surface on a plane wall. The fin is  $20 \text{ mm}$  long, the breadth and thickness being  $200 \text{ mm}$  and  $2 \text{ mm}$  respectively. Assume negligible heat loss from the fin tip. Take  $\theta_0 = 200^\circ\text{C}$ ,  $h = 15 \text{ W/m}^2\text{K}$  and  $K = 45 \text{ W/mK}$ . (10 Marks)

OR

- 4 a. Show that the temperature history of a cooling body with negligible internal resistance is given by,  $\frac{\theta}{\theta_0} = e^{\left(\frac{-hA}{\rho C_p V}\right)t}$ . (10 Marks)  
b. A ball of  $60 \text{ mm}$  diameter at  $600^\circ\text{C}$  is suddenly immersed in controlled medium at  $100^\circ\text{C}$ . Calculate the time required for the ball to obtain a temperature of  $150^\circ\text{C}$ . Assume  $K = 40 \text{ W/mK}$ ,  $\rho = 800 \text{ kg/m}^3$ ,  $C_p = 500 \text{ J/kgK}$ ,  $h = 20 \text{ W/m}^2\text{K}$  for the ball. (10 Marks)

### Module-3

- 5 a. Using dimensional analysis, show that  $Nu = f(Gr, Pr)$  for natural convection. (12 Marks)  
b. Find the heat loss from both sides of a hot square plate  $50 \text{ cm} \times 50 \text{ cm}$  at  $100^\circ\text{C}$ , exposed to atmosphere at  $20^\circ\text{C}$ , if the plate is kept vertical. Use the following relation.

$$Nu = 0.13(Gr.Pr)^{\frac{1}{3}} \rightarrow \text{vertical position at } 60^\circ\text{C}, \text{ take } \rho = 1.06 \text{ kg/m}^3, K = 0.028 \text{ W/mK}$$

$$\gamma = 18.97 \times 10^{-6} \text{ m}^2/\text{s}, C_p = 1.008 \text{ kJ/kg-K}.$$

(08 Marks)

1 of 2

Important Note : 1. On completing your answers, compulsorily draw diagonal cross lines on the remaining blank pages.  
2. Any revealing of identification, appeal to evaluator and/or equations written eg. 42+8 = 50, will be treated as malpractice.

OR

- 6 a. Define the following dimensionless numbers and mention their significance, writing relevant equations:
- Prandtl number
  - Reynolds number
  - Nusselt number
  - Stanton number
- (10 Marks)
- b. Air at 27°C is moving at 0.3 m/s across a 100 W electric bulb at 127°C. If the bulb is approximated by a 10 cm diameter and 1 m high cylinder, estimate the heat transfer rate and percentage of power lost due to convection. At 77°C, take  $Pr = 0.697$ ,  $k = 0.03 \text{ W/mK}$ ,  $\gamma = 2.08 \times 10^{-5} \text{ m}^2/\text{s}$ . (10 Marks)

**Module-4**

- 7 a. Derive an expression for LMTD for a parallel flow heat exchanger. State the assumptions made. (10 Marks)
- b. A heat exchanger is required to cool 55000 kg/h of alcohol from 66°C to 40°C using 40000 kg/h of water entering at 5°C. Assuming parallel flow, calculate
- the exit temperature of water
  - heat transfer
  - surface area required.
- Take  $u = 580 \text{ W/m}^2\text{K}$ ,  $C_p(\text{alcohol}) = 3760 \text{ J/kgK}$ ,  $C_p(\text{water}) = 4180 \text{ J/kgK}$ . (10 Marks)

OR

- 8 a. Sketch and explain different regimes of boiling mechanism. (10 Marks)
- b. Air free saturated steam at 85°C and pressure of 57.8 kPa condenses on the outer surface of 225 horizontal tubes of 1.27 cm OD arranged in a 15×15 array. Tube surface is maintained at 75°C. Find the total condensation rate per m length of the tube bundle.

$$\text{Take : } h_m = 0.725 \left[ \frac{g \rho_l^2 h_{fg} K_c^3}{\mu_l (T_v - T_w) DN} \right]$$

where  $N = \text{Number of tubes}$ ;  $D = \text{Tube diameter}$ ,

Physical properties of water at film temperature of 80°C,  $K_1 = 0.668 \text{ W/m}^2\text{C}$ ,

$\mu_l = 0.355 \times 10^{-3} \text{ kg/m-s}$ ,  $h_{fg} = 2309 \text{ kJ/kg}$ ,  $\rho_l = 974 \text{ kg/m}^3$ .

(10 Marks)

**Module-5**

- 9 a. Define the following laws of radiation, write appropriate relations. Discuss both of them :
- Stefan-Boltzman law
  - Kirchoff's law.
- (12 Marks)
- b. Assuming the sun to be a black body, calculate the surface temperature of the sun and emissive power of the sun's surface. Determine the maximum monochromatic emissive power, taking maximum radiation intensity from sun at  $\lambda = 0.52 \mu$ . (08 Marks)

OR

- 10 a. Define :
- Intensity of radiation
  - Solid angle
  - Lambert's law.
- (12 Marks)
- b. Two large parallel plates are at 1000 K and 800 K. Determine the heat exchange per unit area when (i) The surfaces are black (ii) The hot surface has an emissivity of 0.9 and cold, 0.6. (08 Marks)

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