

CBCS SCHEME

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15MT33

Third Semester B.E. Degree Examination, Dec.2018/Jan.2019 Mechanics of Materials

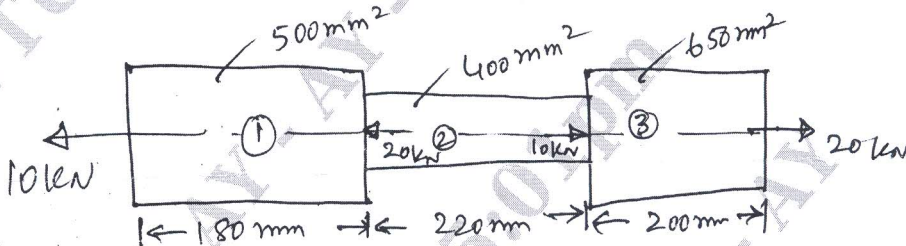
Time: 3 hrs.

Max. Marks: 80

Note: Answer any FIVE full questions, choosing ONE full question from each module.

Module-1

- 1 a. Define the following:
 - i) Poisson's ratio
 - ii) Factor of safety
 - iii) Hooke's law
 - iv) Young's modulus (04 Marks)
- b. Mention the assumption made in the theory of simple stress and strain and derive the equation $dL = \left(\frac{PL}{AE}\right)$. (04 Marks)
- c. Determine (i) Stress induced in each position and (ii) Net deformation in the bar. Take $E = 200 \text{ GPa}$ for stepped bar subjected to varying loads as shown in Fig,Q1(c).



Fig,Q1(c)

(08 Marks)

OR

- 2 a. Derive an expression for the total extension of the tapered circular bar across section of dia 'd₁' and 'd₂', when it is subjected to an axial pull of load 'P'. (08 Marks)
- b. Derive an expression for relation between E, G and μ as $E = 2G(1 + \mu)$. (08 Marks)

Module-2

- 3 a. Define principal stress and principal plane. (02 Marks)
- b. The direct stresses acting at a point in a strained material as shown in Fig.Q3(b). Find the normal, tangential and the resultant stresses on a plane 30° to the plane of major principle stress. Find the obliquity of the resultant stress also.

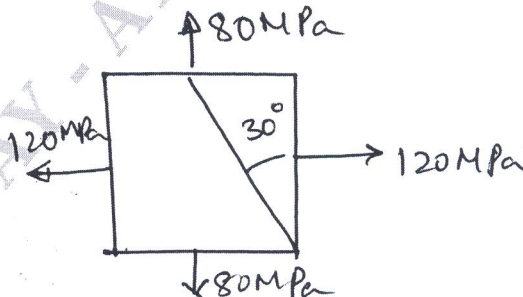


Fig.Q3(b)

(06 Marks)

- c. A plane element is subjected to stresses as shown in Fig.Q3(c). Determine principle stress, maximum shear stress and their planes. Sketch the planes determined.

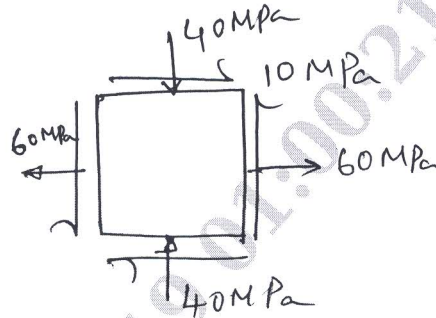


Fig.Q3(c)

(08 Marks)

OR

- 4 a. Derive an expression for circumferential and longitudinal stress for thin cylinder subjected internal pressure 'P'. (08 Marks)
 b. Derive Lames equation for thick cylinder. (08 Marks)

Module-3

- 5 a. Explain shear force, bending moment, hogging moment, sagging moment, point of contraflexure. (05 Marks)
 b. Draw shear force and bending moment diagrams for beam shown in Fig.Q5(b).

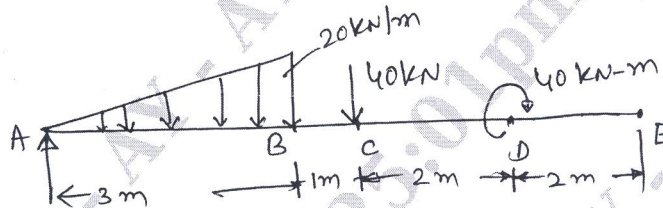


Fig.Q5(b)

(11 Marks)

OR

- 6 Draw the SFD and BMD for the beam shown in Fig.Q6. Find the point of contraflexure.

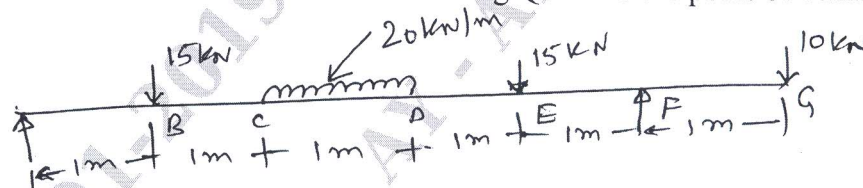


Fig.Q6

(16 Marks)

Module-4

- 7 a. Prove that relation $\frac{M}{I} = \frac{\sigma}{y} = \frac{E}{R}$. (10 Marks)
 b. Show that maximum shear stress of rectangular section is 1.5 times the average shear stress. (06 Marks)

OR

- 8 a. Derive an expression $M = EI \frac{d^2y}{dx^2}$. (10 Marks)
- b. Derive an expression for maximum deflection in a cantilever beam subjected to a point load at free end. (06 Marks)

Module-5

- 9 a. Derive general equation for torsion $\frac{T}{J} = \frac{G\theta}{L} = \frac{\tau}{R}$. (10 Marks)
- b. Explain the following:
- Torsional strength
 - Torsional rigidity
 - Torsional flexibility
 - Section modulus of solid shaft
 - Section modulus of hollow shaft
 - Rigidity modulus
- (06 Marks)

OR

- 10 a. Derive Euler's buckling load for column with its both end hinged. (10 Marks)
- b. A hollow cast iron whose outside dia is 200 mm and thickness 20 mm is 4.5 m long and is fixed at both ends. Calculate the safe load by Rankines formula using a FOS of 2.5. Find the ratio of Euler's to Rankines loads. Take $E = 1 \times 10^5$ MPa, Rankine constant = 1/1600 and $\sigma_y = 550$ MPa. (06 Marks)
